Effect of surface topology and wettability on impacting droplet dynamics at moderate Reynolds numbers

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Abstract

The present work aims to investigate the effect of surface topology and wettability on the impacting droplet dynamics at different flow conditions. A multiphase lattice Boltzmann method (LBM) is employed for the simulation of interfacial dynamics. Firstly, the results obtained based on the present method for some benchmark two-phase flow problems are validated. Then, three surface topologies, including a flat substrate, semicircular cavity, and semicircular bump, are considered to get insight into the physical treatment of the impacting droplet. The present study shows that although the surface topology affects the spreading and rebounding processes of the impacting droplet, the hydrophilicity plays a significant role in the final form of the liquid phase and dictates a similar treatment for all the studied topologies. Considering different sizes for the bump, it is found that the shape of the droplet deforms almost the same immediately after the impaction for all the bump sizes and the spreading process is not affected by the wettability. However, the receding dynamics is significantly affected by the bump size and the wetting condition of the bump surface. It is found that the contact time is minimized by increment the bump size and hydrophobicity of the surface.

Keywords: Impacting droplet, surface topology, wettability effect, multiphase lattice Boltzmann method, Allen-Cahn equation

Nomenclature

С	fluid concentration	t	time
c_s	speed of sound	u	velocity vector
E_{0}	bulk energy	\boldsymbol{x}	position
e	particle velocity	α	lattice direction index
F	total force	β	constant parameter
\mathbf{F}_{p}	pressure force	Γ	hydrodynamic equilibrium function
$\mathbf{F}_{\mathbf{b}}$	body force	θ	contact angle
\mathbf{F}_{μ}	viscous force	κ	coefficient related to surface tension
\mathbf{F}_{s}	surface tension force	μ	chemical potential
G	gravitational acceleration	ξ	interface thickness
g	momentum distribution function	ρ	density
h	concentration distribution function	σ	surface tension coefficient
Μ	Mobility	τ	relaxation time
p	Pressure	φ	Order parameter
R	Radius	ω	weighting coefficient

1- Introduction

There are numerous applications for impacting droplet dynamics on a solid surface, e.g. coating, anti-icing, painting, and spray cooling [1]. Therefore, the studying of droplet impact on substrates is an active field of research and a matter of importance from the physical and applied points of view. Droplet impact on flat surfaces has been extensively studied through experiments and computational techniques [2-4]. Recently, Ouetzeri-Santiago et al. [3] have discussed on the measurement of the dynamic contact angle and the spreading diameter of impacting droplets on a flat surfac. Wei and Thoraval [4] have also numerically study the dynamics of an air-in-liquid compound drop impacting onto a flat surface. However, understanding the physics of the droplet impingement on different surface topologies has recently gained significant attention due to its complex impact hydrodynamics and astounding industrial applications [5-7]. Ezzatneshan and Khosroabadi [5] have presented the effect of textures on a flat surface on the droplet dynamics. Dalgamoni and Yong [6] elucidated the physics of droplet impact on spherical surfaces at different flow conditions. García Pérez et al. [7] employed the LBM approach to Investigations of water droplet impact and freezing on a cold substrate. The ability to control the dynamics of an impacting droplet by changing the surface wettability and geometry in the micro and nano scales can affect many aspects of applications. In various applications such as increasing the efficiency of condensation heat transfer or energy harvesting by jumping droplets, geometric structures can be used for droplet manipulation to control its dynamic behavior. Many types of surface topologies have been suggested for droplet manipulation, e.g. groove types [8, 9]. In this case, some experimental measurements [10-12] and numerical predictions [13-17] have been made in the literature in the past decade.

Recently, Zhifeng et al. [18] have studied a water droplet impaction on ridged superhydrophobic surfaces at varied Weber numbers. They have illustrated the droplet spreading dynamics and revealed the law on the maximum axial spreading coefficient. Huang et al. [19] have adopted a three-dimensional multi-relaxation-time pseudopotential LBM to study the double droplets impact on a wettability-patterned surface. Zhengkai et al. [20] have also implemented a LBM to study on the reducing of the contact time of a droplet impacting on a superhydrophobic substrate via adding a macro-wire or a rectangular ridge. Radhakrishnan et al. [21] have presented the impact dynamics of water droplets at the laser-patterned superhydrophobic surface. They observed finger formation at the start of the spreading and small droplets formed at the tip of fingers which detached from the rim during the retraction. The attention of the aforementioned works has been on the studying of droplet dynamics on a flat or a spherical geometry in different

flow conditions. However, to the best of authors' knowledge, the effect of different surface topologies on the impacting droplet dynamics is not thoroughly considered in the literature. The aim of the present work is to represent a comparative study for the effect of surface topology on the characteristics and pattern of an impacting droplet during the spreading, bouncing, and the rebounding process by a computational model based on the lattice Boltzmann method (LBM).

The LBM is a promising numerical technique for studying complex interfacial dynamics in multiphase flows [22-24]. The implementation of this mesoscopic method is more efficient for imposing wettability properties [25], curved and complex solid walls [26-29], and additional physical properties for special fluid flow systems [20, 30, 31] in comparison with the conventional numerical methods based on the Navier-Stokes equations. In the present work, a multiphase LBM based on the Allen-Cahn (A-C) equation [32] is used to study the effect of the surface topology on the impacting droplet dynamics at different wetting conditions. The shape of the surface geometry, the size of the topology, and the contact angle of the surface are considered as effective parameters. Herein, the simulation of micron-sized droplet dynamics is carried out in which the effect of gravity is stated to be insignificant. In this case, inertia, capillarity, and viscous dissipation play a dominant role in the dynamics of impact.

2- Governing Equations

In the present work, the LBM with A-C equation is used for the simulation of a two-phase flow system. This numerical technique contains two equations for the particle distribution functions, namely h for the prediction of the interfacial dynamics, and g for the calculation of the hydrodynamic properties of the fluid flow. The discretized form of the prior one can be written as follows [32]:

$$h_{\alpha}(\mathbf{x} + \mathbf{e}_{\alpha}\delta t, t + \delta t) = h_{\alpha}(\mathbf{x}, t) - \frac{h_{\alpha}(\mathbf{x}, t) - \overline{h}_{\alpha}^{eq}(\mathbf{x}, t)}{\tau + 0.5} + \delta t \frac{[1 - 4(\phi - \phi_{0})^{2}]}{\xi} \omega_{\alpha} \mathbf{e}_{\alpha} \cdot \frac{\nabla \phi}{|\nabla \phi|}$$

$$\tag{1}$$

In this equation, h_{α} is the phase-field distribution function in the position x, time t, and direction α . The vector \mathbf{e}_{α} is the microscopic velocity of particles, ω_{α} is the weighting factor, δt represents time step, τ is the relaxation time, and ξ in the interfacial thickness that is set to be 4 grid points in the present study. The order parameter ϕ is used to track the interface and $\phi_0 = 0.5(\phi_L + \phi_H)$ indicates the location of the interface by assuming two extreme values, ϕ_L and ϕ_H

, in the bulk of the light and heavy fluids, respectively. The equilibrium phase-field distribution function \bar{h}_a^{eq} is given as:

$$\overline{\mathbf{h}}_{\alpha}^{\mathrm{eq}} = \mathbf{h}_{\alpha}^{\mathrm{eq}} - \frac{1}{2} \delta \mathbf{t} \frac{\left[1 - 4(\phi - \phi_0)^2\right]}{\xi} w_{\alpha} \mathbf{e}_{\alpha} \frac{\nabla \phi}{|\nabla \phi|}$$
(2)

in which, $h_{\alpha}^{eq} = \phi \Gamma_{\alpha}$, and

$$\Gamma_{\alpha} = \omega_{\alpha} \left[1 + \frac{e_{\alpha} \mathbf{u}}{c_{s}^{2}} + \frac{(e_{\alpha} \mathbf{u})^{2}}{2c_{s}^{4}} - \frac{(\mathbf{u} \mathbf{u})}{2c_{s}^{2}} \right]$$

$$(3)$$

In Eq. (3), u indicates the macroscopic velocity vector and $c = \delta x/\delta t$ is set to be unit. Herein, a 2D lattice with nine velocity directions (D2Q9) is considered for the discretization of the domain, as shown in Fig. 1. The order parameter then can be calculated as:

$$\phi = \sum_{\alpha} h_{\alpha} \tag{4}$$

which is used to define the macroscopic density ρ based on the light fluid density, ρ_L , and heavy fluid density, ρ_H , as

$$\rho = \rho_L + (\phi - \phi_L)(\rho_H - \rho_L) \tag{5}$$

The A-C LBM applied in the present study uses the distribution function g_{α} to determine the hydrodynamic properties of multiphase flow systems, including the velocity and pressure fields, as follows:

$$g_{\alpha}(\mathbf{x} + \mathbf{e}_{\alpha}\delta t, t + \delta t) = g_{\alpha}(\mathbf{x}, t) + \Omega_{\alpha}(\mathbf{x}, t) + F_{\alpha}(\mathbf{x}, t)$$
(6)

The term F_{α} imposes the external force F in the simulation by

$$F_{\alpha}(\mathbf{x}, \mathbf{t}) = w_{\alpha} \delta \mathbf{t} \frac{\mathbf{e}_{\alpha} \cdot \mathbf{F}}{\rho c_{s}^{2}} \tag{7}$$

And Ω_{α} describes the collision operator. Herein, to improve the numerical stability of the present A-C LBM for the simulation of multiphase flows with high-density and high-viscosity ratios, a multiple relaxation time (MRT) operator is employed as follows:

$$\Omega_{\alpha} = -\mathbf{M}^{-1}\hat{\mathbf{S}}\mathbf{M} \left(\mathbf{g}_{\alpha} - \overline{\mathbf{g}}_{\alpha}^{\text{eq}} \right)$$
 (8)

The equilibrium distribution function $\ \overline{g}_{\alpha}^{eq}$ is computed by

$$\overline{g}_{\alpha}^{\text{eq}} = g_{\alpha}^{\text{eq}} - \frac{1}{2} F_{\alpha} \tag{9}$$

where

$$g_{\alpha}^{\text{eq}} = p^* w_{\alpha} + (\Gamma_{\alpha} - w_{\alpha}) \tag{10}$$

And $p^* = p/c_s^2$ is the dimensionless pressure. The orthogonal matrix **M** transfers the distribution function from the physical space to the momentum space and $\hat{\mathbf{S}}$ is a coefficient matrix [33]. The macroscopic pressure and velocity are then calculated as follows:

$$p^* = \sum_{\alpha} g_{\alpha} \tag{11}$$

$$\mathbf{u} = \sum_{\alpha} \mathbf{g}_{\alpha} \mathbf{e}_{\alpha} + \frac{\mathbf{F}}{2\rho} \delta \mathbf{t} \tag{12}$$

In the present study, the force vector \mathbf{F} is decomposed to a summation of the surface tension force \mathbf{F}_{s} , body force \mathbf{F}_{b} , pressure force \mathbf{F}_{p} , and viscous force \mathbf{F}_{μ} [5].

3- Model Validation

The accuracy of the present A-C LBM for the computation of multiphase flow properties is assessed by the simulation of two test cases. The equilibrium state of a droplet on a curved surface is considered to evaluate the capability of the A-C LBM for the prediction of the contact angle imposed on the surface and the flow characteristics. Also, the spreading dynamics of an impacting droplet on a flat surface is studied to examine the accuracy of the present numerical technique for the capturing of the interfacial dynamics. A rectangular domain with the grid size of 200×300 is used for the simulation of the equilibrium state of the droplet on a cylinder. The diameter of the droplet and cylinder is set to be D=2R=80 and $D_c=80$, respectively. The solutions are performed at the viscosity ratio 100 and density ratio 1000 with different contact angles.

The quantitative comparison of the equilibrium contact angles obtained on a cylinder surface by the present A-C LBM with those computed by the theoretical prediction based on the Young's law [5] is represented in Fig. 2. It can be seen that the present results are in excellent agreement with the analytical data which confirms the accuracy and efficiency of the A-C LBM employed for resolving the morphology of the droplet in interaction with the curved solid wall for the wide range of wetting conditions. Figure 3-(a) demonstrates the variation of the dimensionless height

of the droplet after reaching the equilibrium condition, H_{max}/R , obtained based on the present A-C LBM in comparison with that reported in the literature [23] based on the numerical simulation and the analytical definition as

$$\frac{H_{max}}{R} = (1 - \cos\theta) \sqrt{\frac{\pi}{2\theta - \sin 2\theta}} \tag{13}$$

As seen in Fig. 3-(a), the present results are in good agreement with the existing data that shows the A-C LBM has accurately resolved the properties of this two-phase flow system on the curved wall of the cylinder.

The dynamics of an impacting droplet on a flat surface with the contact angle $\theta=90^\circ$ is also studied by employing a rectangular flow domain with 200×400 mesh size at the Reynolds number $Re = \frac{\rho_n u_y D}{\mu_H} = 30$ and the weber number $we = \frac{\rho_n u_y^2 D}{\sigma} = 40$. In these definitions, u_y is the vertical component of the velocity $\mathbf{u} = (0, u_y)$ that is imposed initially on the droplet and set to be $u_y = 0.005$. In this case, the instantaneous variation of the wet surface diameter d is measured and its dimensionless magnitude d/D is compared at the different dimensionless time $t^* = iteration/(D/u_y)$ with that obtained by Gupta and Kumar [34] in Fig. 3–(b). This figure indicates that the results agree well and the present A-C LBM accurately predicts the dynamic pattern of this two-phase flow problem. The impact moment and the spreading process of the droplet on the surface are also presented in Fig. 3–(b) with the density field. It can be seen that some gas bubbles are trapped inside the droplet during the impaction. The good match of the results in this section with those in the literature shows that the implemented numerical technique based on the A-C LBM is capable of the modeling of the droplet dynamics on the flat and curved surfaces with different wetting conditions at high viscosity- and density-ratios.

A theoretical expression supported by experimental observations is proposed by Bakshi et al. [10] for an impacting droplet onto a sphere. Herein, the A-C LBM is verified by Bakshi et al.'s work to ensure that the presented results in the present paper for the droplet dynamics on the curved topologies are accurate enough. In this test case, a droplet with the diameter D=60 is initiated in the $(x_0, y_0) = (150,450)$ inside a rectangular domain with the grid size of 300×600 . A solid circular cylinder with the radius $D_c = 60$ and contact angle of is located in the center of the computational domain at $(x_c, y_c) = (150,300)$. Herein, the density and viscosity ratios are set to be

1000 and 100, respectively. The droplet moves towards the cylinder under the gravitational force with the Bond number of d/D $Bo = \frac{g_y(\rho_H - \rho_L)D^2}{\sigma} = 2.2$. At the instant of impact between the droplet and the cylinder surface, when the south pole of the droplet touches the north pole of the cylinder, the time is set to be t_i and the droplet velocity is measured as U_i . For the present test case, these values are computed as $t_i = 11250$ and $U_i = 0.0116$. Accordingly, the dimensionless time $t^* = (t - t_i) \frac{U_i}{D}$ is defined. Also, the liquid film thickness on the top of the cylinder is normalized as $h^* = h/D$. Figure 4 shows the temporal variation of h^* predicted based on the present A-C LBM in comparison with the theoretical and numerical solutions presented in the literature [10, 23]. As expected from the theoretical solution, the film thickness varies as $h^* = 1 - t^*$ at the early time of the impaction. It can be seen that the present solution based on the A-C LBM is in good agreement with the theoretical and numerical solutions in this period of time. At the intermediate time, the theoretical solution proposes a power law as $h^* = 0.15t^{*-2}$ for the variation of the film thickness. However, the present numerical solution as well as the numerical solution given by Fakhari and Bolster [23] show some deviation from the theoretical relation which can be due to the unrealistic nature of 2D simulations in comparison with 3D experimental observations.

4- Results and Discussions

In the present study, three surface topologies, namely the flat, cavity, and bump, are considered to investigate the effect of the geometry and wettability on the spreading dynamics of an impacting droplet on a solid wall. Figure 5 shows the topology and flow parameters of the flat, cavity and bump geometries. Herein, the instantaneous variation of the droplet height along the centerline of the domain, H, and the arc-length of the wetted surface, L, are measured during the spreading process after the impaction. A rectangular computational domain with the size of $2L_0 \times L_0$ is used for all the geometries where $L_0 = 200$. As shown in Fig. 5, a droplet with the radius $R = \frac{L_0}{5}$ is placed at $x = \frac{L_0}{2}$, so that the bottom edge of the drop is tangential to the solid surface. For the cavity and bump cases, a semicircle with the radius of $R_s = 2R$ is considered as the hole and bump, respectively. The present simulations are performed at We = 20, Re = 100 and 900, and imposing different wetting conditions with the contact angles of $\theta = 30^\circ, 60^\circ$ and 90° .

Figure 6 shows the results obtained based on the present A-C LBM for the impacting droplet dynamics on the flat surface. The instantaneous spreading process is demonstrated by the density contours at different contact angles. As observed in this figure, the droplet is deformed immediately after impact on the surface and a liquid lamella spreads with almost the same form for all the contact angles at Re=100. However, at Re=900, a significant increment in the impaction momentum affects the spreading of the liquid lamella in competition with the surface energy corresponding to the considered contact angles. Some tiny entrapped bubbles can be seen in the results presented in Fig. 6 which is due to the gas phase being physically trapped between the droplet and the surface at the impaction moment [35-37]. The capturing and prediction of the entrapped bubble dynamics during the spreading process, also at the equilibrium state demonstrate the capability of the present computational technique for studying the droplet dynamics at different flow conditions.

Figure 7 indicates the present results obtained for the impacting droplet into the cavity at the same conditions used in the previous figure. Obviously, the liquid lamella moves from the center of the cavity along the surface and is propelled upward due to the curvature of the solid wall. In contrast with the flat surface, the cavity shape prevents the spreading droplet to form a thin liquid layer on the solid wall. Figure 8 presents the spreading dynamics of the impacting droplet on a semicircle bump at different flow conditions considered in the present study. Similar to the flat and cavity surface topologies, this figure also shows that the form of the droplet after the impaction moment and the spreading of the liquid lamella on the bump are almost the same for all the contact angels at Re = 100.

The dimensionless arc-length of the wetted surface, L/R, and the droplet height along the centerline of the domain, H/R, are measured to quantitatively compare the impacting droplet dynamics on the flat, cavity, and bump surfaces. Figure 9 demonstrates the comparison of these parameters for the impacting droplet at We = 20 and Re = 100 with the contact angles of $\theta = 30^{\circ},60^{\circ}$ and 90°. The results presented for H/R in this figure shows that the rebounding process of the impacting droplet occurs earlier for the cavity and flat topologies, respectively, in comparison with that on the bump. Indeed, the concave shape of the cavity prevents the liquid lamella to widely spread after impaction. Consequently, the interfacial tension leads to a quick rebound and finally a higher height for the liquid phase is measured into the cavity in comparison with the two other topologies studied. As expected, the instantiates increment of the wet area length L/R

after the droplet impacts on the surface is obvious in Fig. 9 for all the considered surface topologies.

In Fig. 10, the mechanism of the spreading process on the flat, cavity, and bump surfaces is demonstrated by the velocity vectors around the droplet at We = 20, Re = 100 and $\theta = 90^{\circ}$. As shown in this figure, the convex shape of the bump causes less deviation to the vertical impaction momentum in comparison with the flat and cavity topologies. Consequently, the droplet reaches to the maximal possible spreading on the bump until the inertia of the droplet completely diminished. However, the redirected impaction momentum due to the concave shape of the cavity surface prevents the liquid lamella to spread as that observed on the flat and bump surfaces.

The quantitative comparison of the dimensionless arc-length of the wetted surface, L/R, and the droplet height along the centerline of the domain, H/R, is also performed for the impacting droplet dynamics at Re=900. The results obtained for the flat, cavity, and bump surfaces are presented in Fig. 11. Firstly, it can be seen that the duration of the spreading process and receding of the impacting droplet are increased by the increment of the Reynolds number from Re=300 to 900, regardless of the surface topology and contact angle value. Indeed, the increment of the Reynolds number causes the reduction of the viscous damping effect of the liquid against the inertial force of the impaction momentum. Therefore, the liquid lamella tends to widely spread on the surface which leads to a higher value for L/R and lower value for H/R at Re=900 in comparison with those computed at Re=300.

From Figs. 9 and 11, it can be seen that at the low Reynolds number Re=100, the magnitude of H/R after impaction tends to an almost the same value of $|H/R|\approx 2$ for the considered geometries. However, at the moderate Reynolds number Re=900, the decrement rate of H/R is fast for more hydrophilic surface with $\theta=30^{\circ}$ and 60° . In these cases, the final magnitude of |H/R| for the bump geometry is twice its value for flat and cavity geometry. At $\theta=90^{\circ}$, the variation of |H/R| value is almost the same for three geometries. This result shows that the resistance of the surface again the droplet spreading is dominant in comparison with the surface topology. In other words, at higher contact angles, the amount of spreading can be affected by the hydrophobicity of the surface, not its geometry.

The effect of bump size on the impacting droplet dynamics is also investigated at We = 20 and Re = 100. Herein, the contact time of the droplet with the bump surface is also measured at the

range of bump radius $R_s = 1.5R - 3R$ and the contact angle $\theta = 30^{\circ} - 120^{\circ}$. Figure 12 shows the sequences of droplet deformation on the bump targets during the impacting process. It can be seen that the shape of the droplet deforms almost the same for all the considered bump sizes immediately after the impaction ($t^* = 0.15$ and 0.78). From Fig. 12-(a), it can be observed that the liquid lamella covers the whole surface of the small bump with radius $R_s = 1.5R$ and also spreads on the bottom wall around the bump. The hydrophilic property of the bottom wall, in this case, prevents the liquid lamella to recede. With the increment of the bump size in Fig. 12-(a), the spreading lamella does not contact the bottom wall around the bump and tends to recede after the spreading process. The decrement of the surface curvature with the increment of the bump size leads to a larger contact area between the liquid lamella and the surface. Therefore, after reaching the equilibrium state, the maximum height of the liquid phase has been reduced with the increment of the bump radius.

The present results in Fig. 12-(b) and -(c) demonstrate that the impacting droplet exhibits distinct receding behavior at higher contact angles. At $\theta = 90^{\circ}$, despite a similar deformation process on the bump with different sizes, the droplet on the large bump reaches the equilibrium state faster than on the small bump. A completely different dynamics can be observed in Fig. 12-(c) for the droplet receding process on the hydrophobic bump surface at $\theta = 120^{\circ}$. It can be seen that the spreading is restricted immediately after impaction and the droplet then rebounds and separates from the bump surface, despite the bump size.

A similar sensitivity study is performed to investigate the effect of cavity size on the dynamics of impacting droplet at We = 20 and Re = 300. In this case, the cavity radius is set to be $R_s = 1.5R - 2.5R$ and the contact angle is considered $\theta = 30^\circ$. Figure 13 shows the sequences of droplet deformation inside the cavity during the impacting process. It can be seen that the droplet fills the cavity after impaction at $R_s = 1.5R$ and no significant dynamics are then observed. However, with the increment of the cavity diameter to $R_s = 2R$ and 2.5R, the impacting droplet tends to spread on the cavity surface and forms liquid lamella. Although the shape of the droplet deformation is almost the same for the cavity sizes of $R_s = 2R$ and 2.5R, the increment of the cavity size leads to a larger contact area at the similar flow condition (see Fig. 13-(b)). Fig. 13-(b) demonstrates that the spreading arc length reaches a constant value faster at the cavity size $R_s = 1.5R$. However, at the period of time that arc length is almost constant for this case ($r^* > 0.7$), the droplet height exhibits oscillation which is due to the trapping all the liquid volume in a small hole. As the size of the cavity increases, the drop spreads more on hydrophilic surface of the

cavity (higher arc length) and reaching the constant arc length is delayed. Due to the spreading of the liquid lamella on the cavity surface, the liquid height oscillation is not significant for the cavity sizes of $R_s = 2R$ and 2.5R.

Additionally, the dimensionless total contact time $t_{\rm ref}^*$ is measured based on the present solution for the impacting droplet on the hydrophobic bump surface with $\theta = 120^{\circ}$ and 150° in the range of bump size $\frac{R_s}{R} = 1.5, 2, 2.5$ and 3. Figure 14 shows the magnitude of the contact time between the droplet and the bump surface before the rebound. The reduction of the contact time with the increment of the hydrophobicity of the bump surface is obvious in this figure that is consistent with the data reported in the literature [38, 39]. Also, it is found that the bump size affects the contact time at a specific wettability condition. The present results in Fig. 14 demonstrate that the contact time is inversely related to the bump size so that the minimum contact time is obtained for the largest bump size considered in the present study. The reducing of the contact time is crucial in several industrial applications, e.g. to prevent the ice formation on a wing by developing anti-icing surfaces in aerospace engineering. The present work indicates that the computational tool developed based on the A-C LBM is effective and accurate enough for the prediction of flow characteristics in such applications.

5- Conclusion

In the present study, the effect of the surface topology and wettability on the impacting droplet dynamics is investigated by using a computational technique based on the A-C LBM. The model validation against the numerical and experimental data presented in the literature confirms the accuracy of the A-C LBM for the prediction of complex interfacial dynamics. The flat, semicircular cavity and semicircular bump surface topologies are considered to study the flow pattern of the impacting droplet on different surface topologies. The present study shows that although the surface topology affects the dynamics of the impacting droplet during the spreading and rebounding processes, the hydrophilicity of the surface plays a significant role in the final form of the liquid phase and dictates a similar treatment for all the studied topologies. Also, considering different sizes for the bump, it is found that the shape of the droplet deforms almost the same immediately after the impaction for all the bump sizes and the spreading process is not affected by the wettability of the surface. However, the receding dynamics is significantly affected by the bump size and the wetting condition of the bump surface. Additionally, it is

found that the contact time is minimized by the increment of the bump size and the hydrophobicity of the surface.

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Figures

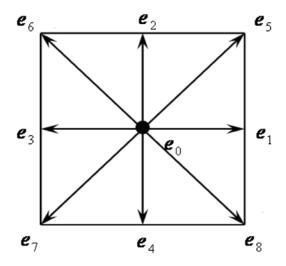


Figure 1. D2Q9 lattice model used for discretization of particle velocity in LBM.

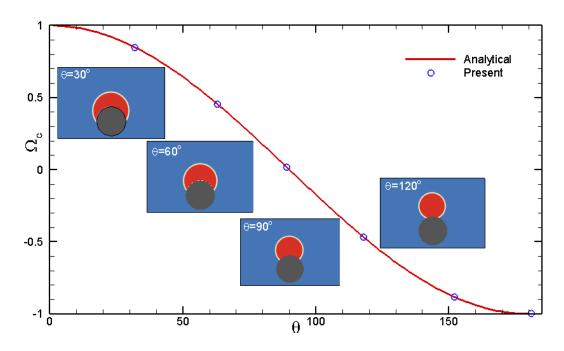


Figure 2. Results obtained by present A-C LBM for equilibrium state of a droplet on a cylinder surface at different wetting conditions, and equilibrium contact angle θ as a function of wetting potential Ω_c in comparison with analytical solution based on Young's law.

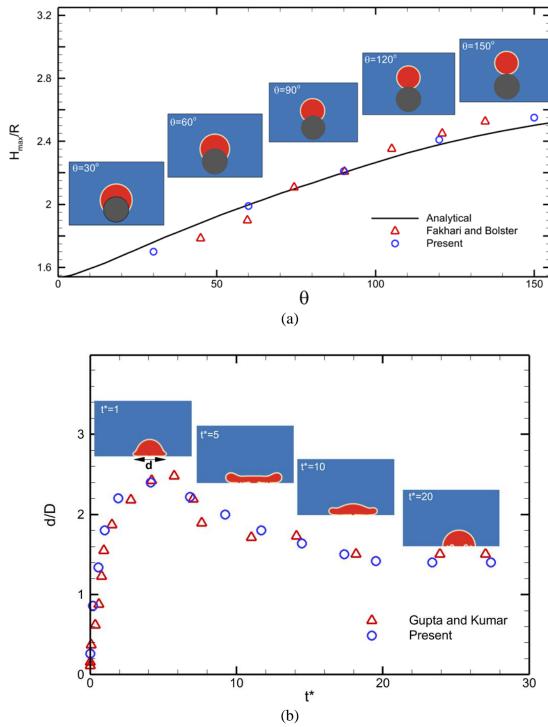


Figure 3. Comparison of present results with available data for (a) droplet equilibrium state on a cylinder at different wetting conditions, and (b) instantaneous evolution of spread factor for impacting droplet on a flat surface at Re = 30 and we = 40.

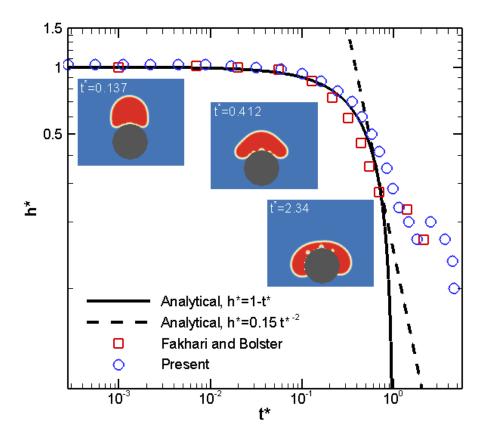


Figure 4. Comparison of instantaneous evolution of dimensionless film thickness h^* for an impacting droplet onto a cylinder obtained based on present A-C LBM with available analytical and numerical solutions at Bo = 2.2 and $\theta = 40^\circ$.

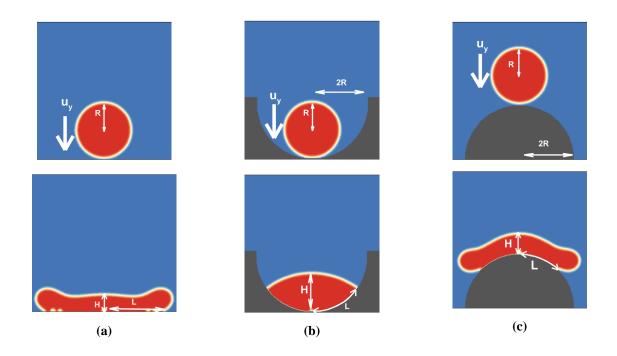


Figure 5. Surface topology and parameters used in present work for study of droplet impact on (a) flat, (b) cavity, and (c) bump geometries.

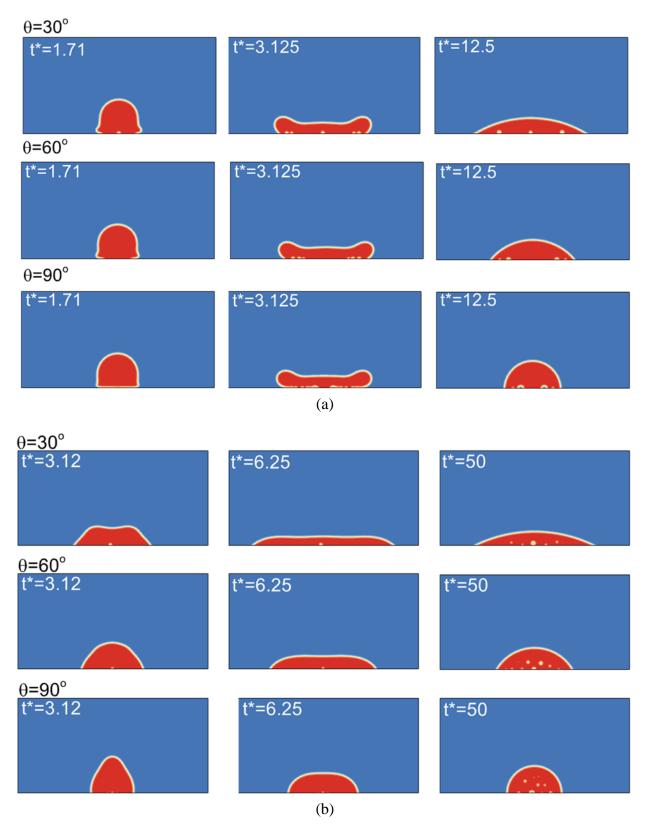


Figure 6. Instantaneous spreading process of impacting droplet on flat surface shown by density contours with We=20, $\theta=30^\circ$, 60° and 90° at (a) Re=100, and (b) Re=900.

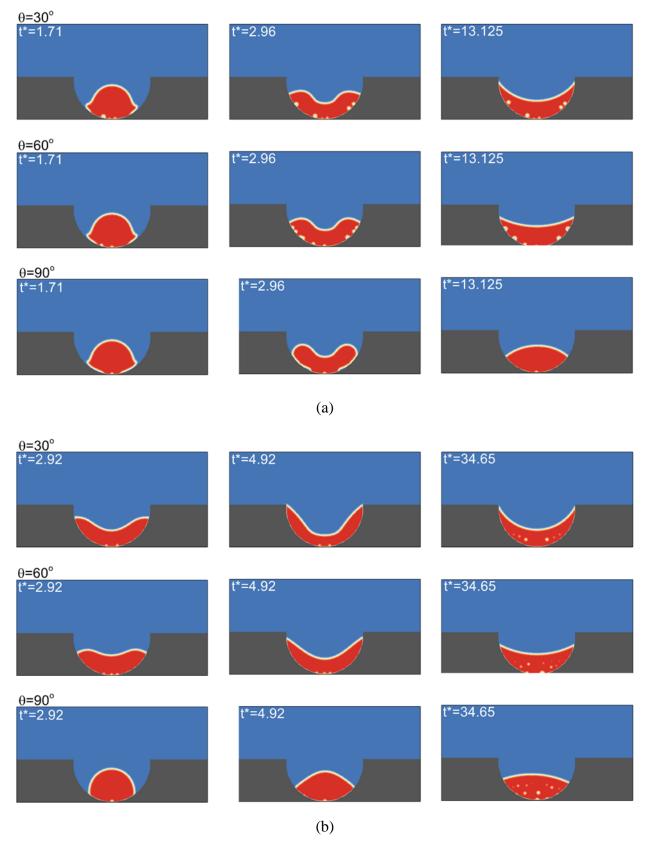


Figure 7. Instantaneous spreading process of impacting droplet in semicircle cavity shown by density contours with We = 20, $\theta = 30^{\circ}$, 60° and 90° at (a) Re = 100, and (b) Re = 900.

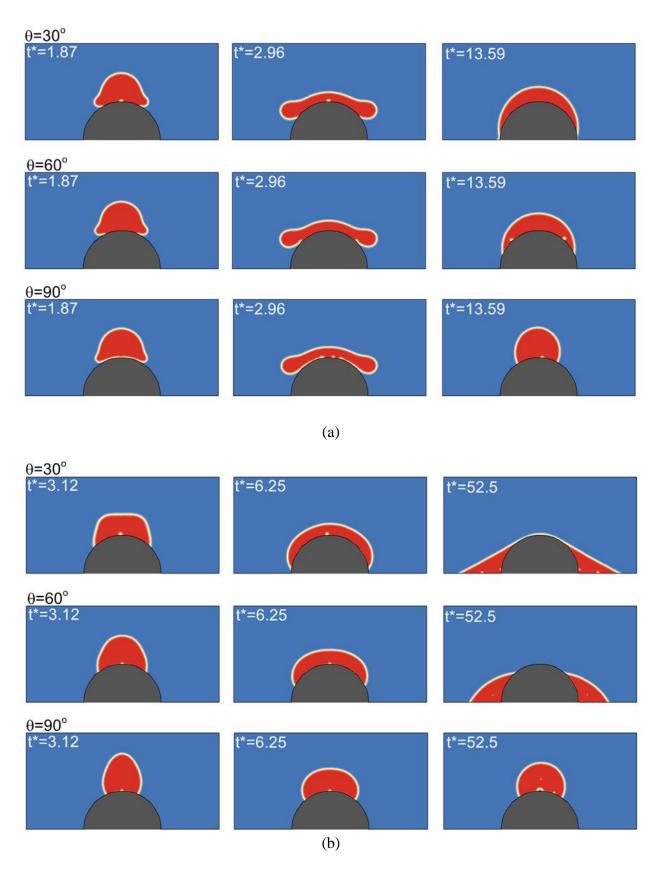
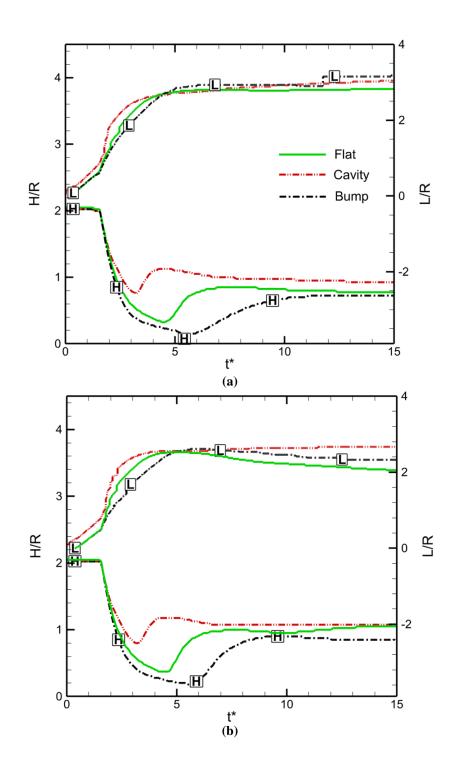


Figure 8. Instantaneous spreading process of impacting droplet on semicircle bump shown by density contours with We = 20, $\theta = 30^{\circ}$, 60° and 90° at (a) Re = 100, and (b) Re = 900.



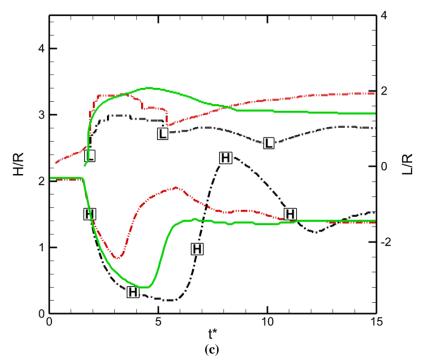


Figure 9. Comparison of dimensionless arc-length of wet area L/R (symbol L) and droplet height H/R (symbol H) during spreading process of impacting droplet on different surface topologies at We=20, Re=100, and (a) $\theta=30^\circ$, (b) $\theta=60^\circ$, (c) $\theta=90^\circ$.

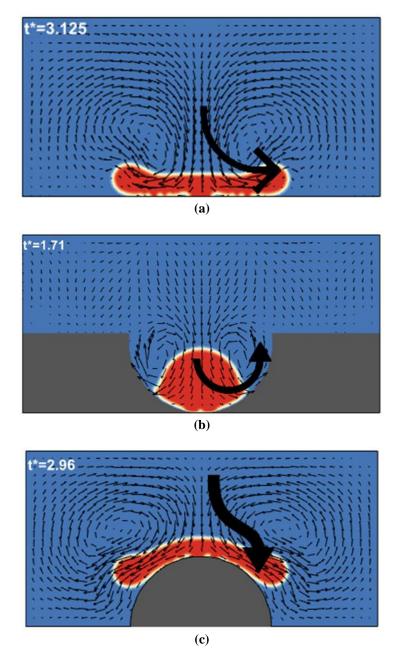
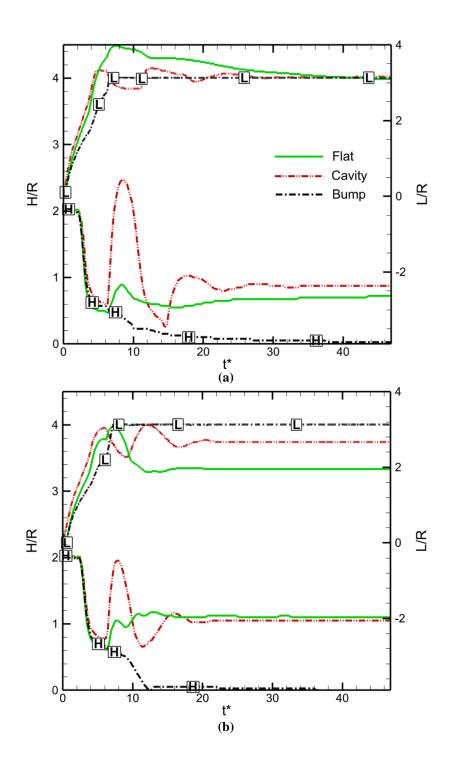


Figure 10. Velocity vectors formed during spreading process around impacting droplet at We = 20, Re = 100, and $\theta = 90^{\circ}$ on (a) flat, (b) cavity, and (c) bump topologies.



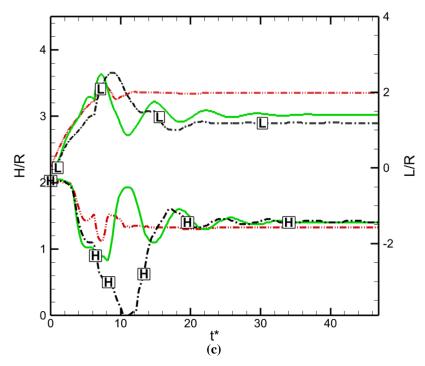


Figure 11. Comparison of dimensionless arc-length of wet area L/R (symbol L) and droplet height H/R (symbol H) during spreading process of impacting droplet on different surface topologies at We=20, Re=900, and (a) $\theta=30^\circ$, (b) $\theta=60^\circ$, (c) $\theta=90^\circ$.

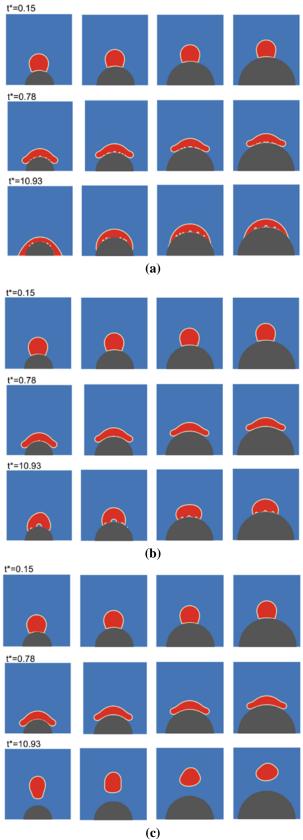


Figure 12. Instantaneous spreading process of impacting droplet on semicircle bump with radius of 1. 5R, 2R, 2. 5R and 3R (from left to right) shown by density contours with We = 20, Re = 100 at (a) $\theta = 30^{\circ}$, (b) $\theta = 90^{\circ}$ and (c) $\theta = 120^{\circ}$.

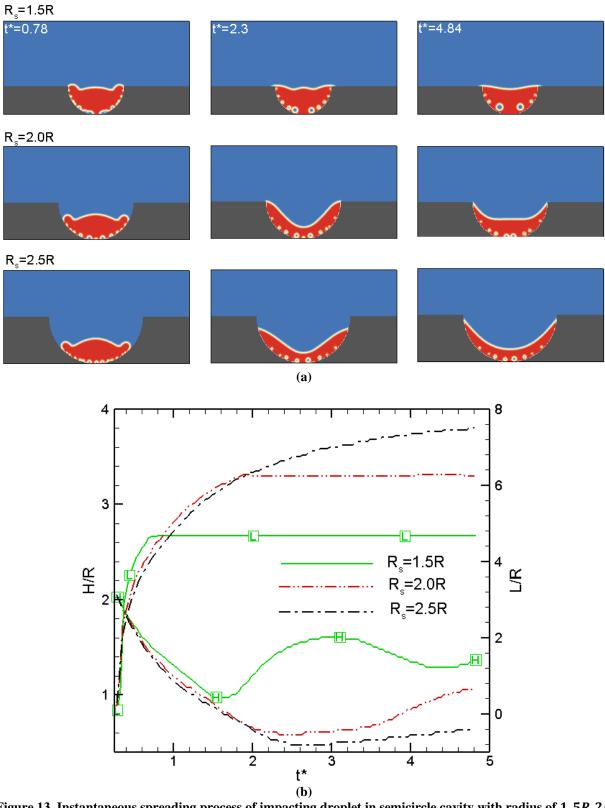


Figure 13. Instantaneous spreading process of impacting droplet in semicircle cavity with radius of 1.5R, 2R and 2.5R, (a) density contours, and (b) dimensionless arc-length of wet area L/R (symbol L) and droplet height H/R (symbol H) during spreading process at We = 20, Re = 300 and $\theta = 30^{\circ}$.

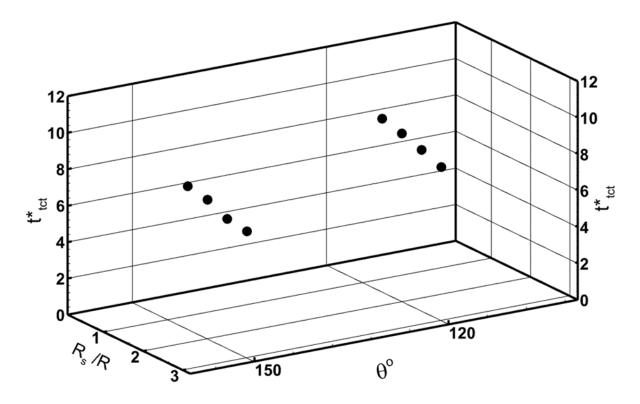


Figure 14. Total contact time computed for impacting droplet on semicircle bump at We = 20, Re = 100, and contact angle of $\theta = 120^{\circ}$ and 150° in a range of bump size (R_s/R) .

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